

MINIATURE MONO RAIL



www.rollon.com

About Rollon



Development of global business

- 1975 Parent company, Rollon S.r.I., founded in Italy
- **1991** Founding of Rollon GmbH in Germany
- 1995 Expansion of headquarters to new 4,000 m² factory Assembly starts in Germany Quality management certified to ISO 9001
- 1998 Rollon B.V. in the Netherlands and Rollon Corporation in the USA are founded
 Expansion of German branch to new 1,000 m² plant
- 1999 Founding of Rollon S.A.R.L. in France Environmental management certified to ISO 14001
- 2000 Rollon s.r.o. founded in Czech Republic
- 2001 Expansion of headquarters to new 12,000 m² manufacturing plant
- 2007 Restructuring of the GmbH and alignment of production in Germany to customer-specific adaptations Takeover of the assets of a manufacturer of linear rail systems
- 2008 Expansion of sales network in Eastern Europe and Asia

Continual expansion and optimization of the portfolio

Founded in 1975, Rollon manufactured high-precision linear roller bearings for the machine tool industry. Early on, Rollon started manufacturing linear bearings based on the bearing-cage design. In 1979, the Compact Rail self-aligning linear bearings joined the Telescopic Rail industrial drawer slides and Easy Rail linear bearings and became the basis of the strong foundation on which the company is building upon today. Continuing optimization of these core products still remains one of the most important goals at Rollon. The development of the patented Compact Rail linear bearing, which uses different proprietary rail profiles and highprecision radial ball bearing sliders, enables the compensation of height and angle mounting defects in applications, and is only one example of the continuing efforts to innovative the development of our existing product families. In the same manner, we continually introduce innovative new product familiesdisplaying our continuing product development and optimization in the industry. These include:

- 1994 Light Rail full and partial extension telescopic in lightweight design
- 1996 Uniline belt driven linear actuators
- 2001 Ecoline economical aluminum linear actuators
- 2002 X-Rail inexpensive formed steel linear guides
- 2004 Curviline curved monorail profile rail guide with roller carriages
- 2007 Monorail miniature sizes and full sized

Each further innovation of our linear bearings is built upon the our extensive knowledge of the nine product families in production today as well as on the current market demands. Rollon is the ultimate linear technology for any application needs.

Content

1 Product explanation	
Mono Rail Miniature Profile Rails	4
2 Technical data	
Performance characteristics and remarks	6
Load capacities	7
3 Product dimensions	
Standard width	8
Large width	9
4 Technical instructions	
Precision	10
Preload, Lubrication	11
Friction, Seal	13
Loading	14
Service life	16
Installation instructions	17

Ordering key Ordering key with explanations and hole pattern

Portfolio

Product explanation

Mono Rail Miniature are profile rails with two-row recirculating balls



The running grooves are ground in gothic arch profile and have a contact angle of 45° so that the same load capacity is guaranteed in all principle directions. Use of large steel balls enables high load and moment capacities despite limited available space.

The most important characteristics:

- Unique ball return circulation
- Integrated ball reverse for improved running properties and increased speeds
- Corrosion resistant
- Mono Rail Miniature profile rails have stable and constantly uniform displacement resistance and require a low breakaway force
- Mono Rail Miniature profile rails are equipped with efficient end seals on both sides of the carriage
- High system rigidity

Preferred areas of application of the Mono Rail Miniature product family:

- Medical technology
- Semi-conductor and electronic industry
- Construction and machine technology
 - (e.g., Pick-and-Place multi-axis systems)

Standard width

Compact technology and high performance in its smallest structural shape.



Large width

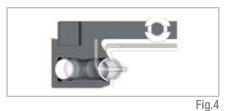
Wide miniature profile rails, with a compact size, allow the acceptance of higher forces and moments. Especially suited for single rail applications.



Unique ball return circulation

The return channels and redirection for the stainless steel balls are made completely out of plastic. The efficient design principle reduces contact between the balls and the metal body. This in turn reduces the generation of noise du-

ring operation. The well-thought-out lubricating principle with lubricant reservoir integrated in the ball recirculation increases the time between lubrication intervals.



Integrated redirection

During the carriage movements, the plastic end caps are subjected to permanent shocks caused by the continuous change of direction of movement. These shock loads have a critical influence on the running properties and speed. Automation and production demands of modern industry call for high operating speeds. The integrated ball redirection, i.e., the direct connection of ball redirection and body, creates an optimum solution for the miniature profile rails.

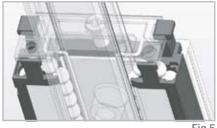
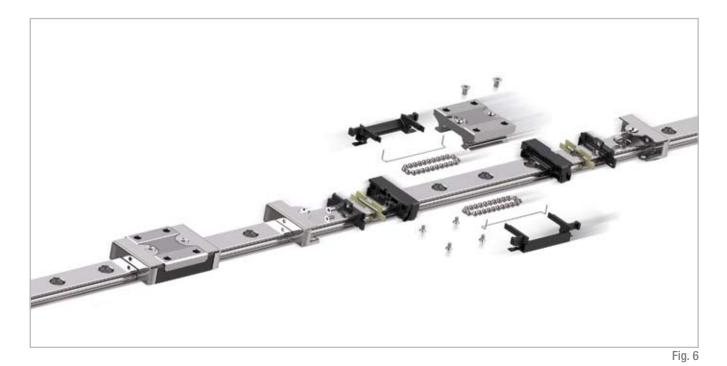


Fig.5

Technical data



Performance characteristics:

- Available sizes of standard width: 7, 9, 12, 15
- Available sizes of large width: 9, 12, 15
- More sizes on request
- Max. operating speed:
 - 3 m/s (118 in/s) (depending on application)
- Max. acceleration:
 250 m/s² (9,844 in/s²) (depending on application)
- Temperature range: -40 °C to +80 °C (-40 °F to +176 °F), temporary temperature peaks are possible up to +100 °C (+212 °F)
- Available single rail lengths up to 1,000 mm (39.37 in)
- Three preload classes: V₀, V_s, V₁
- Three precision classes: P, H, N
- Gothic arch profile with 45° contact angle for the same load capacity in all principle directions
- Corrosion resistant steel

Remarks:

- Combining rails is possible (joining)
- Alternative rail fixing possible on request

Load capacities

Standard width

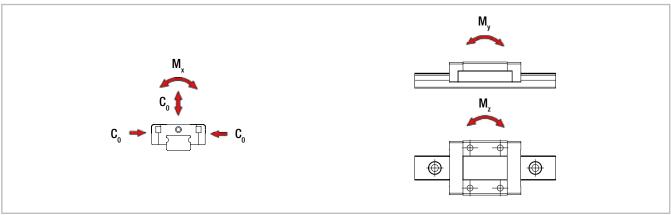


Fig. 7

Туре	Load ca [1	pacities V]	Sta	Static momen [Nm]						
	dyn. C ₁₀₀	stat. C ₀	M _x	M _y	Mz					
MR07MN	890	1400	5.2	3.3	3.3					
MR09MN	1570	2495	11.7	6.4	6.4					
MR12MN	2308	3465	21.5	12.9	12.9					
MR15MN	3810	5590	43.6	27	27					
					Tab. 1					

Large width

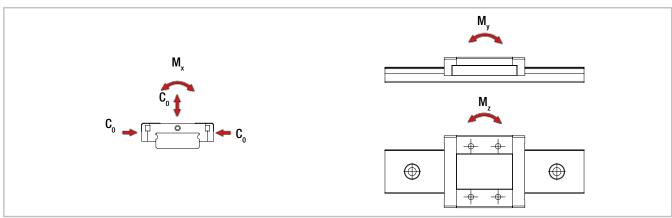


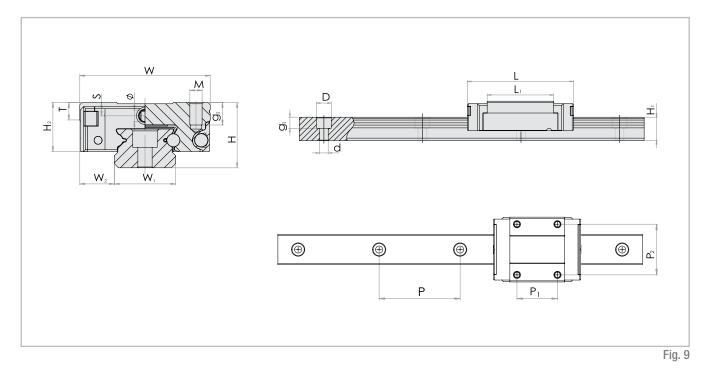
Fig. 8

Туре	Load ca [N	-	Sta	Static moments [Nm]					
	dyn. C ₁₀₀	stat. C ₀	M _x	M _y	Mz				
MR09WN	2030	3605	33.2	13.7	13.7				
MR12WN	3065	5200	63.7	26.3	26.3				
MR15WN	5065	8385	171.7	45.7	45.7				

www.rollon.com

Product dimensions

Standard width



Туре	System [mm]						
	Η	W	W ₂	H ₂			
MR07MN	8	17	5	6.5			
MR09MN	10	20	5.5	7.8			
MR12MN	13	27	7.5	10			
MR15MN	16	32	8.5	12			
				Tab. 3			

Туре		Slider [mm]										Ra [mi					
	L	P ₂	P ₁	Μ	g ₂	L,	т	S	Ø	Weight [kg]	W ₁	H,	Ρ	d	D	g ₁	Weight [kg/m]
MR07MN	23.7	12	8	M2	2.5	14.3	2.8	1.6	1.1	0.008	7	4.7	15	2.4	4.2	2.3	0.215
MR09MN	30.6	15	10	М3	3.0	20.5	3.3	2.2	1.3	0.018	9	5.5	20	3.5	6	3.5	0.301
MR12MN	35.4	20	15	М3	3.5	22.0	4.3	3.2	1.3	0.034	12	7.5	25	3.5	6	4.5	0.602
MR15MN	43.0	25	20	М3	5.5	27.0	4.3	3.3	1.8	0.061	15	9.5	40	3.5	6	4.5	0.93

Tab. 4

Large width

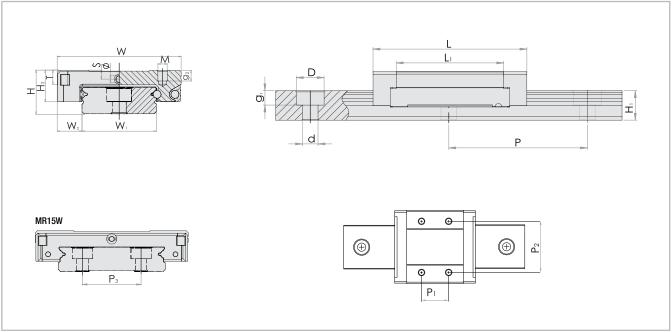


Fig. 10

Туре		System [mm]							
	Н	W	W_2	H ₂					
MR09WN	12	30	6	8.6					
MR12WN	14	40	8	10.1					
MR15WN	16	60	9	12					
				Tab. 5					

Туре	Slider [mm]											Rail [mm]						
	L	P ₂	P ₁	М	g ₂	L,	Т	S	Ø	Weight [kg]	W ₁	H	Р	P ₃	d	D	g ₁	Weight [kg/m]
MR09WN	39.1	21	12	М3	3	27.9	4	2.6	1.3	0.037	18	7.3	30	-	3.5	6		0.94
MR12WN	44.4	28	15	М3	3.5	31.0	4.5	3.1	1.3	0.065	24	8.5	40	-	4.5	8	4.5	1.472
MR15WN	55.3	45	20	M4	4.5	38.5	4.5	3.3	1.8	0.137	42	9.5	40	23	4.5	8		2.818
																		Tab. 6

Technical instructions

Precision

There are three precision classes to choose from for the Mono Rail Miniature profile rails: Classes P, H, and N are manufactured.

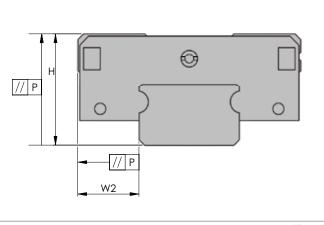


Fig. 11

	Precision classes	Precision P [µm]	High H [µm]	Normal N [µm]
н	Tolerance of height H	± 10	± 20	± 40
ΔН	Permissible height difference of different carriages at the same position on the rail	7	15	25
W ₂	Tolerance of width W_2	± 15	± 25	± 40
ΔW_2	Permissible width difference of different carriages at the same position on the rail	10	20	30

Tab. 7

Running accuracy

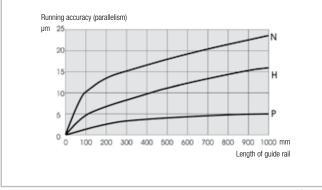


Fig. 12

Preload

The Mono Rail Miniature profile rails are available in the three different preload classes $V_{_{\rm O}},V_{_{\rm S}}$ and $V_{_{\rm I}}$ (see table 8). The preload influences the rigidity, precision and torque resistance and also affects the product service life and displacement force.

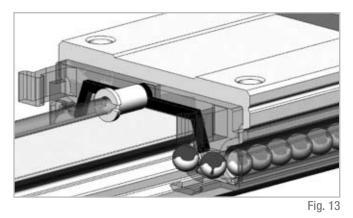
Туре		Preload classes	
.]}0	Small clearance Very quiet running V _o [µm]	Standard Very quiet and precise running V _s [µm]	Small preload High rigidity, vibration reduced, high precision, good load balance V ₁ [µm]
MR07	+4 to +2	+2 to 0	0 to -3
MR09	+4 to +2	+2 to 0	0 to -4
MR12	+5 to +2	+2 to 0	0 to -5
MR15	+6 to +3	+3 to 0	0 to -6
			Tab. 8

Lubrication

Function

The contact points between ball and track are separated from each other by a microscopically thin oil film. The lubrication effects:

- Reduction of friction
- Reduction of wear
- Corrosion protection
- Better thermal distribution and therefore increased of service life



Important instructions for lubrication

- Mono Rail Miniature profile rails must be lubricated for operation.
- The carriage must be moved back and forth during lubrication.
- The lubricant can also be applied to the tracks.
- The lubricant can be injected into the lubrication holes on both sides of the carriage.
- There should be a thin film of lubricant on the rail surface at all times.
- Please inform us in advance if the guides are to be used in acid or base containing environments or in clean rooms.
- Please contact the sales department if the oil lubrication should be used for vertical use of the guide.
- If the stroke is < 2 or > 15 times the carriage length, the lubrication intervals should be more often.

Grease lubrication

When using grease lubrication, we recommend synthetic-oil based lithium grease with a viscosity according to ISO VG 32 to ISO VG 100.

Oil lubrication

We recommend CLP or CGLP synthetic oil conforming to DIN 51517 or HLP to DIN 51524 and a viscosity range conforming to ISO VG 32 to ISO VG 100 for operating temperatures between 0 °C and +70 °C. We recommend a viscosity according to ISO VG 10 for use at low temperatures. For application-specific special lubrication please contact the Rollon Application engineering department.

ISO VG 10	ê	Viscosity of 10 $\frac{mm^2}{s}$	at 40 °C
ISO VG 32	ê	Viscosity of 32 $\frac{mm^2}{s}$	at 40 °C
ISO VG 100	Ê	Viscosity of 100 $\frac{mm^2}{s}$	at 40 °C

Fig. 14

Туре	First lubrication [cm ³]
MR07MN	0.12
MR09MN	0.23
MR12MN	0.41
MR15MN	0.78
	Tab. 9

Туре	First lubrication [cm ³]
MR09WN	0.30
MR12WN	0.52
MR15WN	0.87
	Tah 10

Tab. 10

Lubrication intervals

Operating speed, stroke length and ambient conditions influence the selection of time between lubrication intervals. Establishing a safe lubrication interval is based exclusively on the experienced practiced values determined on site. However, a lubrication interval should not be longer than one year in any case.

Relubrication

- Relubrication of the system must be done before the lubricant used is dirty or shows discolouration.
- An application of approx. 50 % of the quantity used for first lubrication is sufficient for relubrication (see tab. 9f).
- Relubrication is performed at operating temperature.
 During relubrication, the carriage should be moved back and forth.
- If the stroke is < 2 or > 15 times the carriage length, the lubrication intervals should be more often.

Friction

The Mono Rail Miniature profile rails have a low friction characteristic with constant running resistance and low breakaway force.

$\boldsymbol{F}_{m}=\boldsymbol{\mu}\cdot\boldsymbol{F}$	F = load (N) $F_m = friction force (N)$
	Fig. 15

Causes of friction

- Friction of the sealing system
- Friction of the balls with each other
- Friction between balls and redirection
- Rolling resistance of the balls in the gothic arch running grooves
- Resistance of lubricant in the carriage
- Resistance by contamination in the lubricant

Friction with lubricated end seal

Туре	M [N _{max}]	W [N _{max}]	
MR07	0.1	-	
MR09	0.1	0.8	
MR12	0.4	1.0	
MR15	1.0	1.0	
		Tab. 11	

Seal

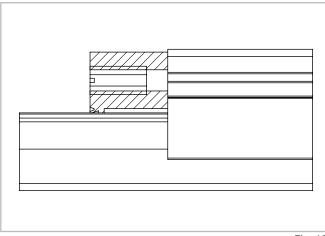


Fig. 16

Mono Rail Miniature profile rails have a coefficient of friction of approx. $\mu = 0.002 \text{ - } 0.003.$

The carriages of the Miniature Mono Rail are equipped with end seals on both sides.

The design of the end seal ensures a good and dust-proof seal. This extends the product service life, reduces the loss of lubricant and guarantees the optimum system lubrication over a long time.

The special design of the stripper allows a low seal resistance and has no negative influence on the running behaviour of the system.

Loading

Static load (P₀) and static moment (M₀)

Permissible static load

The permissible static load of the Mono Rail Miniature profile rail is limited by:

- Static load of each linear guide
- Permissible load of the fixing screws
- Permissible load of all components used in the surrounding construction
- Static safety factor, which is required by the corresponding application

The equivalent static load and the static moment are the largest load, or the largest moment, which are calculated based on formulas 3 and 4.

Static load capacity C₀

The static load capacity C_0 of ball recirculating guides is defined according to DIN 636, Part 2 as the only load which gives a Hertzian stress of 4,200 MPa with the existing lubrication between track and balls in the center of the highest loaded contact surface.

Note: In the loading center, there is a permanent deformation of approx 0.01 % of the ball diameter under this load (according to DIN 636, Part 2).

Static safety factor S₀

When observing the static safety factor $\rm S_0$ the Mono Rail Miniature profile rails allow a permissible operation and high running precision as is required for each application. Calculation of the static safety factor $\rm S_0$: see fig. 17

- S₀ static safety factor
- $\mathrm{C}_{_{\mathrm{O}}}$ static load capacity in loading direction (N)
- P₀ equivalent static load (N)
- M_o static moment in loading direction (Nm)
- M equivalent static moment in loading direction (Nm)

$S_{0} = C_{0} / P_{0}$	Formula 1	Operating conditions	S ₀
$S_0 = M_0 / M$	Formula 2	Normal operation	1 ~ 2
$P_0 = F_{max}$	Formula 3	Loading with vibration or shock effect	2 ~ 3
$M_0 = M_{max}$	Formula 4	High precision and smooth running	≥ 3
			Fig. 17

Dynamic load capacity C

If the dynamic loads work vertically on the last zones with equal size and direction, the calculated service life of the linear guide can theoretically reach 100 km piston travel (as per DIN 636, Part 2).

Combined loads in combination with moments

If both loads and moments work on the profile rails, the equivalent dynamic load is calculated with formula 9. According to DIN 636, Part 1, the equivalent load should not exceed $\frac{1}{2}$ C.

Equivalent dynamic load and speed

With changing load and speed, these must be considered individually since each parameter helps determine the service life.

Equivalent dynamic load

If only the load changes, the equivalent dynamic load can be calculated with formula 5.

Equivalent speed

If only the speed changes, the equivalent speed is calculated with formula 6. If speed and load change, the equivalent dynamic load is calculated with formula 7.

Combined dynamic load

With combined exterior load in an arbitrary angle, the equivalent dynamic load is calculated with formula 8.

$$P = \sqrt[3]{\frac{q_1 \cdot F_1^3 + q_2 \cdot F_2^3 + \dots + q_n \cdot F_n^3}{100}} Formula 5$$

$$\overline{v} = \frac{q_1 \cdot v_1 + q_2 \cdot v_2 + \dots + q_n \cdot v_n}{100} Formula 6$$

$$P = \sqrt[3]{\frac{q_1 \cdot v_1 + q_2 \cdot v_2 + \dots + q_n \cdot v_n}{100}} Formula 6$$

$$P = \sqrt[3]{\frac{q_1 \cdot v_1 + q_2 \cdot v_2 + \dots + q_n \cdot v_n}{100}} Formula 6$$

$$P = \sqrt[3]{\frac{q_1 \cdot v_1 + F_1^3 + q_2 \cdot v_2 \cdot F_2^3 + \dots + q_n \cdot v_n \cdot F_n^3}{100}} Formula 7$$

$$P = \sqrt[3]{\frac{q_1 \cdot v_1 \cdot F_1^3 + q_2 \cdot v_2 \cdot F_2^3 + \dots + q_n \cdot v_n \cdot F_n^3}{100}} Formula 8$$

$$P = \sqrt[3]{\frac{q_1 \cdot v_1 + F_1^3 + q_2 \cdot v_2 \cdot F_2^3 + \dots + q_n \cdot v_n \cdot F_n^3}{100}} Formula 8$$

$$P = \sqrt[3]{\frac{q_1 \cdot v_1 + F_1^3 + q_2 \cdot v_2 \cdot F_2^3 + \dots + q_n \cdot v_n \cdot F_n^3}{100}} Formula 8$$

$$P = \sqrt[3]{\frac{q_1 \cdot v_1 + F_1^3 + q_2 \cdot v_2 \cdot F_2^3 + \dots + q_n \cdot v_n \cdot F_n^3}{100}} Formula 8$$

$$P = \sqrt[3]{\frac{q_1 \cdot v_1 + F_1^3 + q_2 \cdot v_2 \cdot F_2^3 + \dots + q_n \cdot v_n \cdot F_n^3}{100}} Formula 8$$

$$P = \sqrt[3]{\frac{q_1 \cdot v_1 + F_1^3 + q_2 \cdot v_2 \cdot F_2^3 + \dots + q_n \cdot v_n \cdot F_n^3}{100}} Formula 8$$

$$P = \sqrt[3]{\frac{q_1 \cdot v_1 + F_1^3 + q_2 \cdot v_2 \cdot F_2^3 + \dots + q_n \cdot v_n \cdot F_n^3}{100}} Formula 8$$

$$P = \sqrt[3]{\frac{q_1 \cdot v_1 + F_1^3 + q_2 \cdot v_2 \cdot F_2^3 + \dots + q_n \cdot v_n \cdot F_n^3}{100}} Formula 8$$

$$P = \sqrt[3]{\frac{q_1 \cdot v_1 + F_1^3 + q_2 \cdot v_2 \cdot F_2^3 + \dots + q_n \cdot v_n \cdot F_n^3}{100}} Formula 8$$

$$P = \sqrt[3]{\frac{q_1 \cdot v_1 + F_1^3 + q_2 \cdot v_2 \cdot F_2^3 + \dots + q_n \cdot v_n \cdot F_n^3}{100}} Formula 8$$

$$P = \sqrt[3]{\frac{q_1 \cdot v_1 + F_1^3 + q_2 \cdot v_2 \cdot F_2^3 + \dots + q_n \cdot v_n \cdot F_n^3}{100}} Formula 8$$

$$P = \sqrt[3]{\frac{q_1 \cdot v_1 + F_1^3 + q_2 \cdot v_2 \cdot F_2^3 + \dots + q_n \cdot v_n \cdot F_n^3}{100}} Formula 8$$

$$P = \sqrt[3]{\frac{q_1 \cdot v_1 + F_1^3 + q_2 \cdot v_2 \cdot F_2^3 + \dots + q_n \cdot v_n \cdot F_n^3}{100}} Formula 9$$

$$P = \sqrt[3]{\frac{q_1 \cdot v_1 + F_1^3 + q_2 \cdot v_2 \cdot F_2^3 + \dots + q_n \cdot v_n \cdot F_n^3}{100}} Formula 9$$

$$P = \sqrt[3]{\frac{q_1 \cdot v_1 + F_1^3 + q_2 \cdot v_2 \cdot F_2^3 + \dots + q_n \cdot v_n \cdot F_n^3}{100}} Formula 9$$

$$P = \sqrt[3]{\frac{q_1 \cdot v_1 + F_1^3 + q_2 \cdot v_2 \cdot F_2^3 + \dots + q_n \cdot v_n \cdot F_n^3}{100}} Formula 9$$

$$P = \sqrt[3]{\frac{q_1 \cdot v_1 + F_1^3 + q_1 +$$

Fig. 18

Service life

An example of a profile rail or a lot of identical profile rails under the same running conditions, which use ordinary materials with normal manufacturer's quality and operating conditions, can reach 90 % of the calculated service life (as per DIN 636 Part 2). By taking 50 km traverse as a basis, the dynamic load capacity is usually 20 % over the values as per DIN. The relationship between the two load capacities can be seen from formulas 10 and 11.

Calculation of service life

Formulas 12 and 13 are used for calculating the service life, if equivalent dynamic load and average speed are constant.

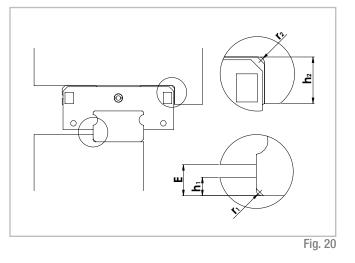
$C_{_{(50)}} = 1,26 \cdot C_{_{(100)}}$	Formula 10	L = service life based on 100,000 (m)
$C_{_{(100)}} = 0,79 \cdot C_{_{(50)}}$	Formula 11	L_h = service life (h) C = dynamic load capacity (N)
$L = (\frac{C_{100}}{P})^3 \cdot 10^5$	Formula 12	P = equivalent dynamic load (N) S = stroke length (m)
$L_{h} = \frac{L}{2 \cdot s \cdot n \cdot 60} = \frac{L}{V_{m}} \cdot (\frac{C_{100}}{P})^{3}$	Formula 13	n = stroke frequency (min $^{-1}$) V _m = average speed (m/min)

Fig. 19

Installation instructions

Shoulder heights and radius of stop edges

Rounding of the stop edges of the surrounding construction should be made so as to avoid contact with the edges of the carriage and the rail. Please observe the following table with the information on the radius and height of the stop surfaces.



Dimensions of the stop edges

Туре	h ₁ [mm]	r _{ımax} [mm]	h ₂ [mm]	r _{2max} [mm]	E [mm]
MR07M	1.2	0.3	2.8	0.3	1.5
MR09M	1.5	0.3	3	0.3	2.2
MR12M	2.5	0.5	4	0.5	3
MR15M	2.5	0.5	4.5	0.5	4

Туре	h ₁ [mm]	r _{ımax} [mm]	h ₂ [mm]	r _{2max} [mm]	E [mm]
MR09W	2.5	0.3	3	0.3	3.4
MR12W	2.5	0.5	4	0.5	3.9
MR15W	2.5	0.5	4.5	0.5	4
					Tab. 13

Tab. 12

Geometric and positional accuracy of the mounting surfaces

Inaccuracies of the mounting surface negatively influence the running accuracy and reduce the service life of the Mono Rail Miniature profile rails. If the inaccuracies of the mounting surfaces exceed the values calculated using formulas 14, 15 and 16, the service life is shortened according to formulas 12 und 13.

Mounting surface

The mounting surface should be ground or milled very finely and have a surface roughness of R_a 1.6.

Reference surface

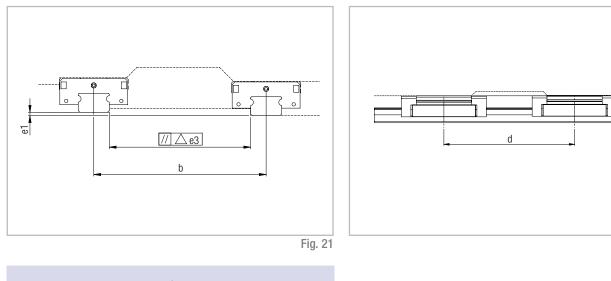
Rail: Both sides of the rails can be used as a reference surface without further marks.

Slider: The reference surface is located across from the running side identified with a notch mark.

www.rolle	on.com

17

Calculation of the positional accuracy



 V_1

f2

10

10

12

10

f3

3

4

6

8 Tab. 14

e1 (mm) = b (mm) \cdot f1 \cdot 10 ⁻⁴	Formula 14
e2 (mm) = d (mm) \cdot f2 \cdot 10 ⁻⁵	Formula 15
e3 (mm) = f3 · 10 ⁻³	Formula 16
	Fig. 23

f3

4

6

8

12

f1

3

4

4

5

 $\mathbf{V}_{0}, \mathbf{V}_{S}$

f2

11

11

13

11

f1

5

5

6

7

Туре

MR07MN

MR09MN

MR12MN

MR15MN

Туре	V ₀ , V _S			V ₁		-
.,,,,,	f1	f2	f3	f1	f2	f3
MR09WN	2	7	6	2	5	4
MR12WN	3	8	8	2	5	5
MR15WN	2	9	11	1	6	7
						Tab. 15

GЗ

Fig. 22

Tightening torque for fixing screws (Nm)

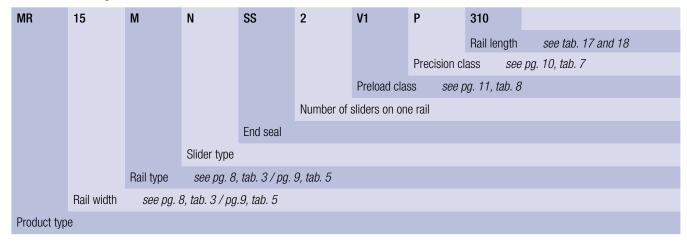
Screw quality 12.9	Steel	Cast iron	Non-ferrous metal
M2	0.6	0.4	0.3
М3	1.8	1.3	1
M4	4	2.5	2
			Tab. 16

Tab. 16

Notes

Ordering key

Rail / slider system

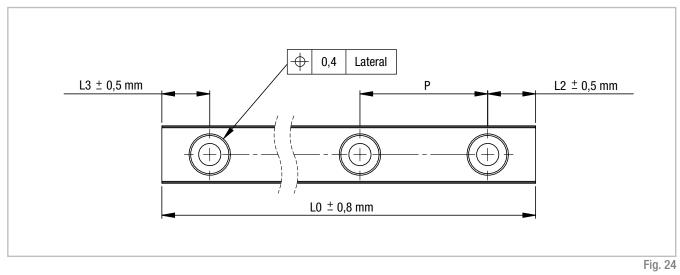


Ordering example: MR15MN-SS-2-V1-P-310

Hole pattern: 15-7x40-15, see fig. 24, tab. 17 / fig. 25, tab. 18

Hole pattern

Standard width



Size	L _{min} [mm]	Hole pitch P [mm]	L ₂ , L _{3min} [mm]	L ₂ , L _{3max} * [mm]	L _{max} [mm]
7	40	15	3	10	
9	55	20	4	15	1000
12	70	25	4	20	1000
15	70	40	4	35	
* does not apply to minimum	(L _{min}) and maximum ra	uil length (L _{max})			Tab. 17

Large width

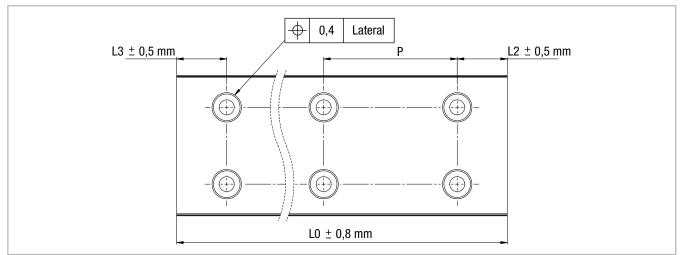


Fig. 25

Size	L _{min} [mm]	Hole pitch P [mm]	L ₂ , L _{3min} [mm]	L ₂ , L _{3max} * [mm]	L _{max} [mm]
9	50	30	4	25	
12	70	40	F	35	1000
15	110	40	5	35	
*	(I) and mailer in the second	() () () () () () () () () () () () () (Tob 10

* does not apply to minimum (L_min) and maximum rail length (L_max)

Tab. 18

Portfolio



COMPACT RAIL

Rugged roller sliders with innovative self adjustment



MONO RAIL

Profile guideways for highest degrees of precision



CURVILINE Curvilinear rails for constant and variable radii



TELESCOPIC RAIL

Smooth-running telescopic linear bearing drawer slides with low deflection under heavy loads



EASY RAIL Compact, versatile linear bearings



X-RAIL

Roller embossed stainless steel profiles for the use in rough environments



UNILINE

Steel-reinforced, belt-driven linear actuators with hardened steel linear bearings and precision radial ball bearing rollers



LIGHT RAIL

Full and partial extension, lightweight drawer slides

Fold out ordering key

To make this product catalog as simple as possible for you to use, we have included the following easy-to-read chart.

Your advantages:

- Description and ordering designations easy to read at one glance
- Simplified selection of the correct product
- Links to detailed descriptions in the catalog



ROLLON S.r.I.

Italy

Via Trieste 26 I-20871 Vimercate (MB) Tel.: (+39) 039 62 59 1 Fax: (+39) 039 62 59 205 E-Mail: infocom@rollon.it www.rollon.it

i o n

ROLLON GmbH

Bonner Strasse 317-319 D-40589 Düsseldorf Tel.: (+49) 211 95 747 0 Fax: (+49) 211 95 747 100 E-Mail: info@rollon.de www.rollon.de

ROLLON S.A.R.L.

Les Jardins d'Eole, 2 allée des Séquoias F-69760 Limonest Tel.: (+33) (0)4 74 71 93 30 Fax: (+33) (0)4 74 71 95 31 E-Mail: infocom@rollon.fr www.rollon.fr

ROLLON B.V.

Ringbaan Zuid 8 6905 DB Zevenaar Tel.: (+31) 316 581 999 Fax: (+31) 316 341 236 E-Mail: info@rollon.nl www.rollon.nl

ROLLON Corporation

101 Bilby Road. Suite B Hackettstown, NJ 07840 Tel.: (+1) 973 300 5492 Fax: (+1) 908 852 2714 E-Mail: info@rolloncorp.com www.rolloncorp.com

All addresses of our global sales partners can also be found in the internet at www.rollon.com

d. The text and images may be